Computer Aided Machine Drawing

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Limits and tolerance

The manufacture of interchangeable parts require precision.

Precision is the degree of accuracy to ensure the functioning of a part as intended.

However, experience shows that it is impossible to make parts economically to the exact dimensions. This may be due to,

- (i) inaccuracies of machines and tools (<u>Machine factor</u>),
- (ii) inaccuracies in measurements and setting the work to the tool (<u>Human factor</u>), and (iii) material requirements (<u>material factor</u>).
- The workman, therefore, must be given some allowable margin so that he can produce an economical part.
- The dimensions of which will lie between two acceptable limits, a maximum and a minimum.

Following are some of the terms used in the limit system:

- *Tolerance*

- The permissible variation of a size is called tolerance.
- It is the difference between the maximum and minimum permissible limits of the given size (difference between upper and lower limits).
- If the variation is provided on one side of the basic size, it is termed as *unilateral tolerance*.
- Similarly, if the variation is provided on both sides of the basic size, it is known as *bilateral tolerance*.

Following are some of the terms used in the limit system:

- <u>Limits</u>

- The two extreme permissible sizes between which the actual size is contained are called limits.
- The maximum size is called the <u>upper limit</u> and the <u>minimum size</u> is called the lower limit.

Following are some of the terms used in the limit system:

- <u>Deviation</u>

It is the algebraic difference between a size (actual, maximum, etc.) and the corresponding <u>basic size</u> (the amount by which dimension differ from the zero line).

Following are some of the terms used in the limit system:

- *Upper Deviation*

It is the algebraic difference between the <u>maximum limit</u> of a size and the corresponding <u>basic size</u>.

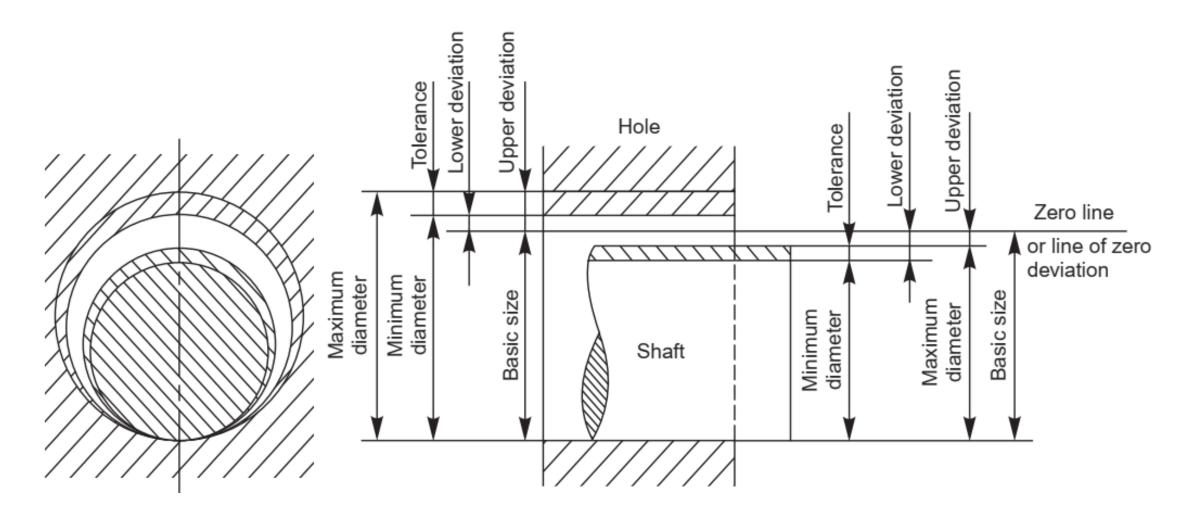
Following are some of the terms used in the limit system:

- Lower Deviation

It is the algebraic difference between the *minimum limit* of the size and the corresponding *basic size*

Following are some of the terms used in the limit system:

- <u>Basic size (nominal size)</u>
- It is determined from design calculations.
- If the strength and stiffness requirements need a 50mm diameter shaft,
- Then 50mm is the basic shaft size.
- If it has to fit into a hole, then 50 mm is the basic size of the hole



Tolerance

In this chapter we concentrates on dimensional tolerance that controls a dimensional feature such as length and diameter and it is usually bounded by upper and lower limits.

It has two types:

- Unilateral tolerance (one side)
- Bi-lateral tolerance (two sides)

The tolerance should be selected to optimize the cost without affecting a part job.

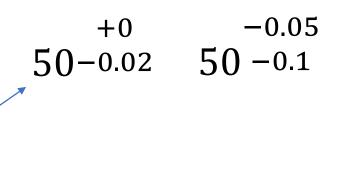
Tolerance

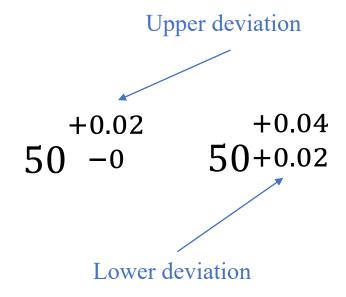
Unilateral tolerance (one side)

The tolerance zone always lie above or below zero line.

Examples

Basic size





Tolerance

Bilateral tolerance (two sides)

The tolerance zone lies above and below the zero line. It is a combination between +ve and –ve deviation.

Examples

$$+0.02$$
 $50-0.02$

$$+0.05$$
 $50 -0.1$

Upper limit is 50.02 Lower limit is 49.98

Fundamental Tolerance

Defines the position of tolerance zone in relation to the zero line (basic size). There are 25 types of fundamental tolerances:

Holes

A, B, C, D, E, F, G, H, Js, J, K, M, N, P, R, S, T, U, V, X, Y, Z, Za, Zb, Zc Shafts

a, b, c, d, e, f, g, h, js, j, k, m, n, p, r, s, t, u, v, x, y, z, za, zb, zc

Excluded letters I, L,O, Q, W

Excluded letters i, l,o, q, w

Fit

Is the degree of tightness between two mating parts.

It is classified into three categories:

- Clearance fit
- Transition fit, and
- Interference fit

Fit

• *Clearance fit:*

The upper limit of shaft is less than the lower limit of hole. The tolerance zone of both shaft and hole allows a clearance between them.

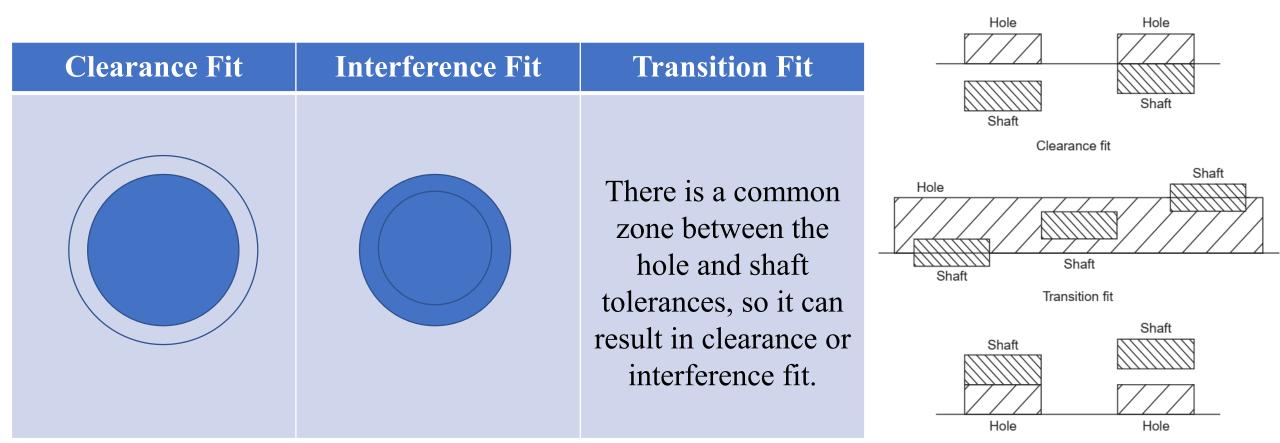
• *Interference fit:*

Upper limit if hole is less than lower limit of shaft. The tolerance zone between shaft and hole allows interference between them.

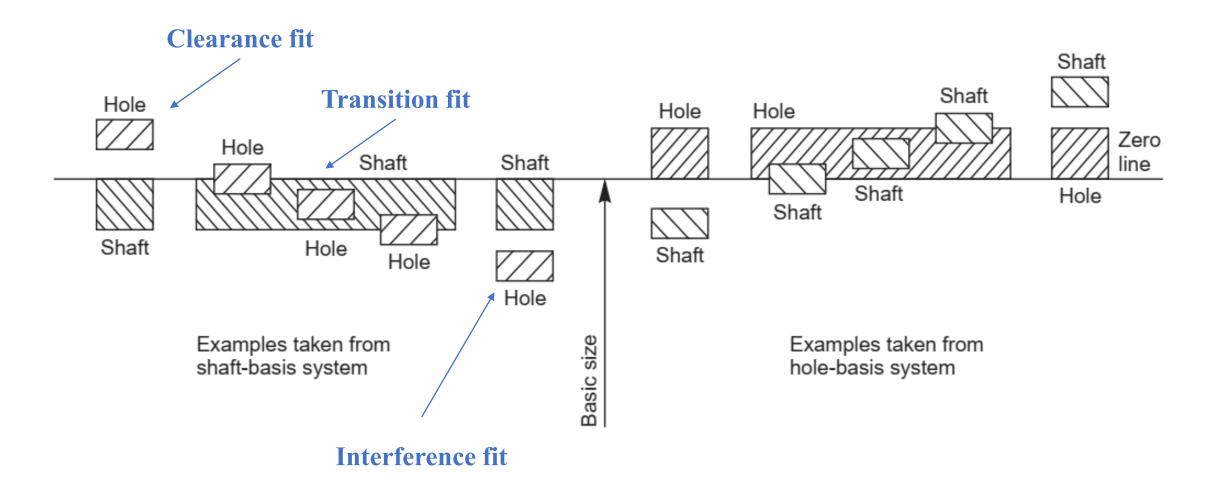
• *Transition fit:*

Tolerance zone of hole overlapping the tolerance zone of shaft.

Fit

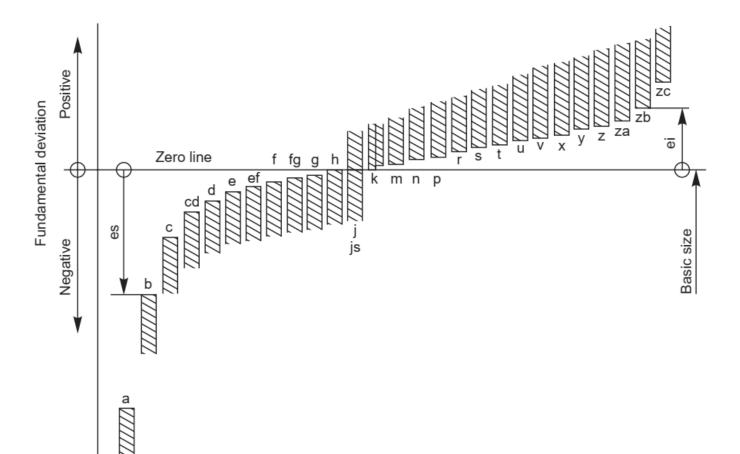


Shaft basis Vs hole basis systems



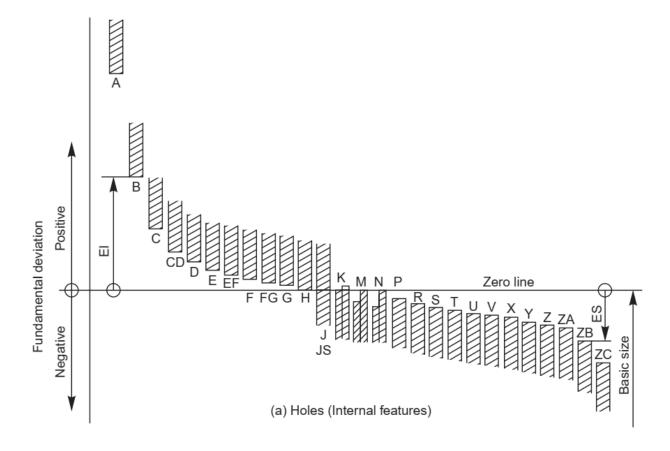
Shaft basis Vs hole basis systems

In shaft basis system of limits and fits fundamental deviation of shaft always will be h



Shaft basis Vs hole basis systems

In hole basis system of limits and fits fundamental deviation of hole always will be H



50 *H*6 *g*7

50 – basic size/ nominal size

H – the fundamental tolerance for hole

6 – hole tolerance grade

g – the fundamental tolerance for shaft

7 – shaft tolerance grade

The H fundamental tolerance for hole refers to hole basis system

50 *H*6 *g*7

mm

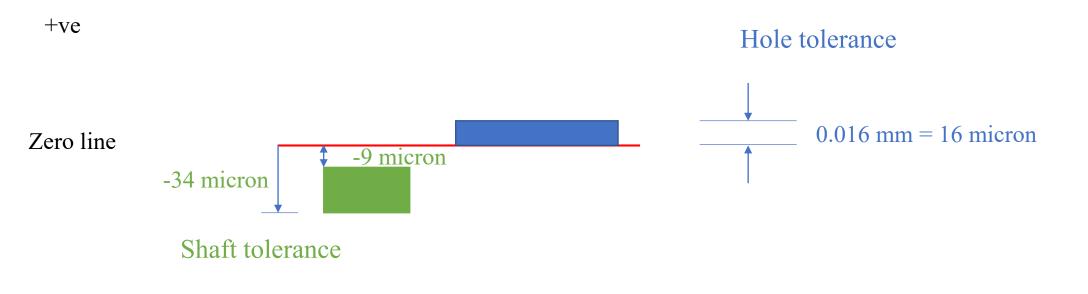
H6 SIZE **H8 H7 H5** H16 H15 H14 H13 H12 H11 H10 H9 **H4 H3** H2 Н1 OVER 0 0.600 0.400 0.250 0.140 0.100 0.060 0.040 0.025 0.014 0.010 0.006 0.004 0.003 0.002 0.001 0.0008 TO 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.0000 OVER 0.750 0.480 0.300 0.180 0.120 0.075 0.048 0.030 0.018 0.012 0.008 0.005 0.004 0.003 0.002 0.0010 TO 6 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.0000 0.000 0.000 0.000 0.000 OVER 0.900 0.580 0.360 0.220 0.150 0.022 0.015 0.0010 6 0.090 0.058 0.036 0.009 0.006 0.004 0.003 0.002 TO 10 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.0000 OVER 10 0.070 0.027 0.018 0.0012 1.100 0.700 0.430 0.270 0.180 0.110 0.043 0.011 0.008 0.005 0.003 0.002 TO 14 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.0000 OVER 14 0.270 1.100 0.700 0.430 0.180 0.110 0.070 0.043 0.027 0.018 0.011 0.008 0.005 0.003 0.002 0.0012 TO 18 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.0000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 **OVER** 18 0.0015 1.300 0.840 0.520 0.330 0.210 0.130 0.084 0.052 0.033 0.021 0.013 0.009 0.006 0.004 0.003 TO 24 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.0000 OVER 24 0.520 0.330 0.210 0.084 0.033 0.021 0.013 0.009 0.0015 1.300 0.840 0.130 0.052 0.006 0.004 0.003 TO 30 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.0000 OVER 30 1.600 1.000 0.620 0.390 0.250 0.160 0.100 0.062 0.039 0.025 0.016 0.011 0.003 0.0015 0.007 0.004 TO 40 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.0000 **OVER** 40 0.390 0.250 0.039 0.025 0.016 0.0015 1.600 1.000 0.620 0.160 0.100 0.062 0.011 0.007 0.004 0.003 TO 50 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.0000 OVER 50 1.900 1.200 0.740 0.460 0.300 0.190 0.120 0.074 0.046 0.030 0.019 0.013 0.008 0.005 0.003 0.0020 65 TO 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.0000

	TIC	
50	Hh	α /
50	<i>H</i> 6	\mathcal{G}

SIZE		g9	g8	g7
OVER	0	-0.002	-0.002	-0.002
ТО	3	-0.027	-0.016	-0.012
OVER	3	-0.004	-0.004	-0.004
ТО	6	-0.034	-0.022	-0.016
OVER	6	-0.005	-0.005	-0.005
то	10	-0.041	-0.027	-0.020
OVER	10	-0.006	-0.006	-0.006
то	14	-0.049	-0.033	-0.024
OVER	14	-0.006	-0.006	-0.006
ТО	18	-0.049	-0.033	-0.024
OVER	18	-0.007	-0.007	-0.007
то	24	-0.059	-0.040	-0.028
OVER	24	-0.007	-0.007	-0.007
то	30	-0.059	-0.040	-0.028
OVER	30	-0.009	-0.009	-0.009
то	40	-0.071	-0.048	-0.034
OVER	40	-0.009	-0.009	-0.009
ТО	50	-0.071	-0.048	-0.034
OVER	50	-0.010	-0.010	-0.010
то	65	-0.084	-0.056	-0.040

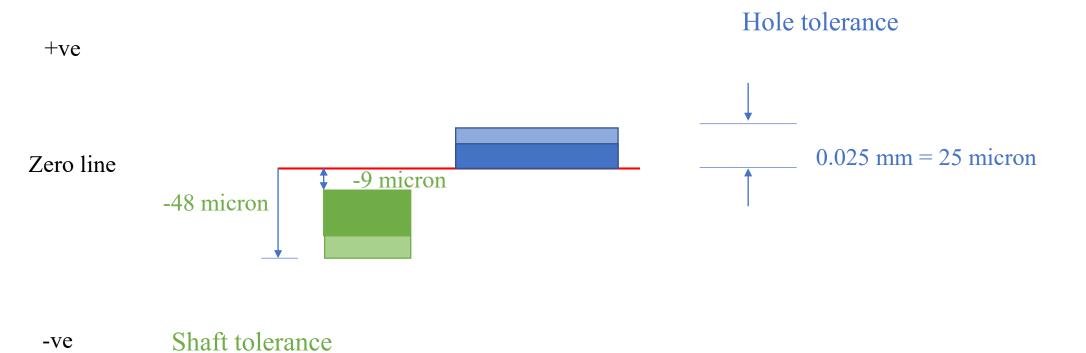
50 *H*6 *g*7

-ve



50 *H*7 *g*8

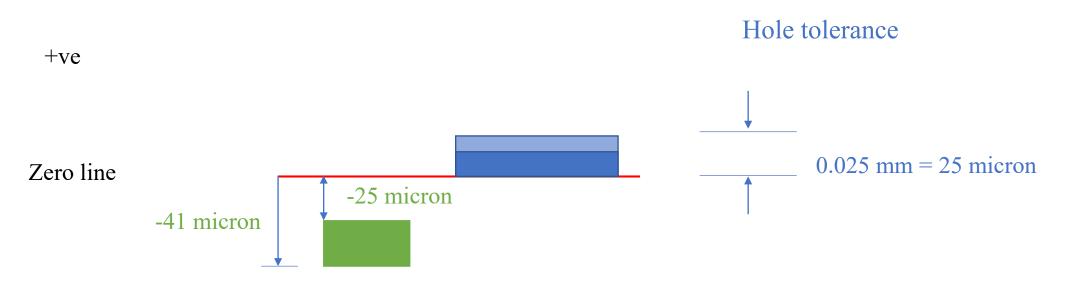
-ve



50 *H*7 *f*6

Shaft tolerance

-ve



All these tolerances are hole basis and represents a clearance fit.

